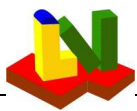


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## Layout

Input values:	1.234	or	1.234
Calculated values:	1.234	or	1.234
Critical values:	1.234	or	1.234
Estimated values:	1.234	or	1.234

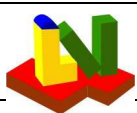


**ASME BPVC VIII-1 2017**  
Example E4.6.1 - E4.6.2 PTB-4-2013

## Summary

Strength  
Calculation  
Software      Program System ATLAS      Version **8.29.1**  
Developed by Lauterbach Verfahrenstechnik  
GmbH  
Certified per  
DIN EN ISO      Certificate Number 01 100  
9001:2015      044763

	LV Soft	ASME	Diff [%]
<b>Example E4.6.1 - Flat Unstayed Circular heads Attached by Bolts</b>			
<b>Required plate thickness t</b>	41,95 mm    1,65 in	1,65 in	0,05%
<b>Example E4.6.2 - Flat Unstayed Non Circular heads Attached by Welding</b>			
<b>Required plate thickness t</b>	17,88 mm    0,70 in	0,70 in	0,09%



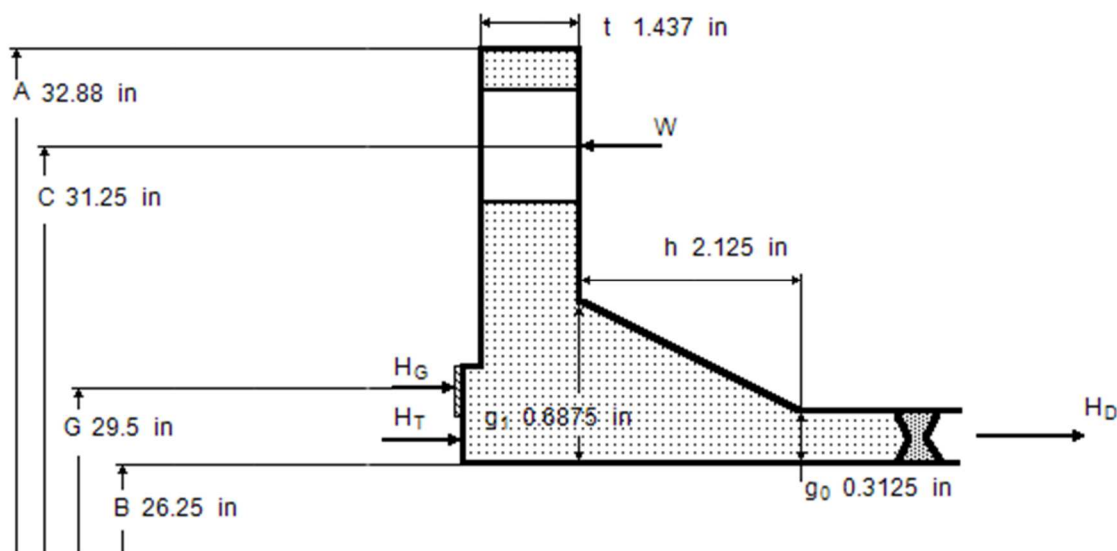
### E 4.16.1 - Bolted flanges - ASME BPVC VIII Division 1 App. 2: 2017

#### Integral Type Flange

##### Design data

Design pressure	$P_D$	135 psi	$= p_D$	135 psi
Hydrostatic head	$D_P$	0 psi	$= D_p$	0 psi
Calculation pressure	$P_0$	135 psi	$= p_0$	135 psi
Calculation temperature			$T_0$	650 °F

#### Flange



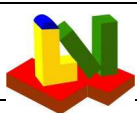
Outside diameter	A	32.88 in	Inside diameter	B	26.25 in
Bolt circle diameter	C	31.25 in	Pipe size	$B_n$	26.25 in
Hub length	h	2.125 in	Flange thickness	t	1.437 in
Large hub thickness	$g_1$	0.6875 in	Small hub thick.	$g_0$	0.3125 in

Material K03504-SA-105--Class:-Size:

Allowable operating stress	$S_{fb}$	17811 psi
Allowable installation stress	$S_{fa}$	20015 psi
Corrosion allowance	$c_2$	0 in
Modulus of elasticity at operation	$E_T$	2.591e+7 psi
Modulus of elasticity at test (20°C)	$E_{20}$	2.92e+7 psi

#### Gasket

Gasket diameter	G	29.5 in
Effective gasket width	b	0.2031 in
Gasket factor	m	3.75
Gasket seating load	y	7600 psi



# ASME BPVC VIII-1 2017

## Example E4.6.1 - E4.6.2 PTB-4-2013

### Bolts

Number	n	44
Root diameter	$d_k$	0.62 in
Nominal diameter	a	0.75 in
Material	G41400-SA-193-B7-Class:-Size:<=64	
Allowable operating stress	$S_b$	24946 psi
Allowable installation stress	$S_a$	24946 psi
Consider bolt spacing correction factor $B_{SC}$ 2-6(7)?	(N=No) Y	(Y/N)
Required operation bolt load	Eq.(1)	$W_{m1}$ <b>111274</b> lbf
Minimum initial bolt load	Eq.(2)	$W_{m2}$ <b>142982</b> lbf
Available cross section of bolts	$A_b$	<b>13.28</b> in <sup>2</sup>
Required cross section	$W_{m1}/S_b$	$A_{m1}$ <b>4.46</b> in <sup>2</sup>
Required cross section	$W_{m2}/S_a$	$A_{m2}$ <b>5.732</b> in <sup>2</sup>
Req. bolt load for gasket seating	Eq.(5)	$(A_m + A_b) \cdot S_a / 2$ W <b>237101</b> lbf
Allowable bolt load	$A_b \cdot S_a$	$W_{all}$ <b>331221</b> lbf
Design (gasket seating =1; max. allowable=2)		1 (1,2)

### Moment

	Force	·	Lever arm	=	Result
$M_D = H_D \cdot h_D$	= 73024 lbf	·	2.156 in	=	13122 lbf·ft
$M_G = H_G \cdot h_G$	= 19049 lbf	·	0.875 in	=	16667 lbf·in
$M_T = H_T \cdot h_T$	= 19202 lbf	·	1.688 in	=	32403 lbf·in
Total operating moment	$M_{01} = M_D + M_G + M_T$			=	206529 lbf·in
Total gasket seating moment, Eq. (6)	$M_{02} = W \cdot (C-G)/2$			=	207464 lbf·in

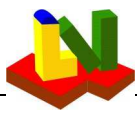
### Stress

Longitudinal	$S_H$	Operation <b>17786</b> psi	Installation <b>17866</b> psi	$\leq$ Allowable	
Ratio	$S_H/S_f$	<b>0.9986</b>	<b>0.8926</b>	$\leq 1.5 \cdot S_f$	Eq.(8)
Allowable stress	$S_f$	17811 psi	20015 psi		
Radial	$S_R$	<b>6157</b> psi	<b>6184</b> psi	$\leq S_f$	Eq.(9)
Tangential	$S_T$	<b>5548</b> psi	<b>5573</b> psi	$\leq S_f$	Eq.(10)
Combination	$(S_H + S_R)/2$	= <b>11971</b> psi	<b>12025</b> psi	$\leq S_f$	
Combination	$(S_H + S_T)/2$	= <b>11667</b> psi	<b>11719</b> psi	$\leq S_f$	
Bolt pitch	$B_S$	<b>2.231</b> in	$\leq$ <b>3.529</b> in	= $B_{Smax}$	Eq.(3)

### Remark

Cross-sectional area of bolts  
Strength condition flange





**Auxiliary values**

$$K = \frac{A}{B} = 1.252$$

$$T = 1.817 \quad (\text{Fig. 2-7.1})$$

$$U = 9.623 \quad (\text{Fig. 2-7.1})$$

$$Y = 8.757 \quad (\text{Fig. 2-7.1})$$

$$Z = 4.518 \quad (\text{Fig. 2-7.1})$$

$$h_0 = \sqrt{B \cdot g_0} = 72.75 \text{ mm}$$

$$F = 0.7677 \quad (\text{Fig. 2-7.2})$$

$$V = 0.1576 \quad (\text{Fig. 2-7.3})$$

$$f = 1 \quad (\text{Fig. 2-7.6})$$

$$d = \left( \frac{U}{V} \right) \cdot h_0 \cdot g_0^2 = 279869 \text{ mm}^3$$

$$e = \frac{F}{h_0} = 0.01055 \text{ 1/mm}$$

$$L = \frac{(t \cdot e + 1)}{T} + \frac{t^3}{d} = 0.9359$$

$$H = 0.785 \cdot G^2 \cdot P \cdot 0.1 = 410239 \text{ N}$$

$$H_D = 0.785 \cdot B^2 \cdot P \cdot 0.1 = 324826 \text{ N}$$

$$H_P = 2 \cdot b \cdot \pi \cdot G \cdot m \cdot P \cdot 0.1 = 84732 \text{ N}$$

$$H_T = H - H_D = 85412 \text{ N}$$

$$W_{m1} = H + H_P = 494970 \text{ N} \quad \text{Eq.(1)}$$

$$W_{m2} = \pi \cdot b \cdot g \cdot y = 636011 \text{ N} \quad \text{Eq.(2)}$$

$$H_G = W_{m1} - H = 84732 \text{ N}$$

$$R = \frac{(C-B)}{2} - g_1 = 46.04 \text{ mm}$$

$$h_D = R + 0.5 \cdot g_1 = 54.77 \text{ mm}$$

$$h_G = \frac{(C-G)}{2} = 22.23 \text{ mm}$$

$$h_T = \frac{(R + g_1 + h_G)}{2} = 42.86 \text{ mm}$$

Bolt pitch

$$B_S = \pi \cdot \frac{C}{n} = 56.67 \text{ mm}$$

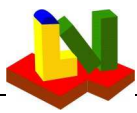
Eq.(3)

$$B_{Smax} = 2 \cdot a + 6 \cdot \frac{t}{(m+0.5)} = 89.63 \text{ mm}$$

For

$$B_S > 2 \cdot a + t$$

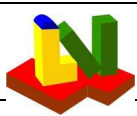
$$\text{Lauterbach Verfahrenstechnik } C_{B_{SC}} = \sqrt{\frac{B_S}{(2 \cdot a + t)}} = 1 \quad \text{Eq.(7)} \quad 17.04.2019$$



**ASME BPVC VIII-1 2017**  
Example E4.6.1 - E4.6.2 PTB-4-2013

KI (=0.3 acc. Table 2-14) = **0.3**

Rigidity criterion: J **0.8339**  $\leq 1.0$



## E 4.6.1 - Unstayed flat heads and covers - ASME BPVC VIII-1 UG-34 & UG-39: 2017

### Circular flat heads and plates with flange moment

#### Design data

Design pressure	$p_D$	135 psi
Hydrostatic head	$D_p$	0 psi
Calculation pressure	$p_0$	<b>135</b> psi
Calculation temperature	$T_0$	650 °F
Design type (Fig. UG-34)	Type	1

#### Gasket

Gasket diameter	$G$	29.5 in
Effective gasket width	$b$	0.2031 in
Gasket factor	$m$	3.7
Gasket seating load	$y$	7600 psi

#### Bolt forces

Gasket seating force $W$ acc. 2-5(e) Eq.(5), AFL	$W_{E1}$	237101 lbf
Lever arm	$h_g$	0.875 in

#### Flat head or plate

Final wall thickness	$t_h$	1.437 in
Wall thickness allowance	$c_1$	0 in
Allowance (corrosion)	$c_2$	0.125 in
Wall thickness without allowances	$t_0$	<b>1.312</b> in
Design diameter	$d$	29.5 in
Joint efficiency	$E$	1

#### Material data

Material	K03504-SA-105--Class:-Size:	
Allowable stress installation	$S_E$	20015 psi
Allowable stress operation	$S_B$	17811 psi

#### Results

Gasket force for min. pressure	$W_{m2}$	<b>142982</b> lbf
Bolting force for installation $\text{MAX}(W_{E1}, W_{m2})$	$W_E$	<b>237101</b> lbf
Bolt force for operation	$W_{m1}$	<b>111020</b> lbf
Design factor	$C$	0.3
Required thickness	$t$	<b>1.526</b> in
Required thickness incl. allowances	$t + c_1 + c_{2 < 7 \text{sub}} >$	<b>1.651</b> in
Minimum required thickness in a groove	$t_m$	<b>0.8171</b> in

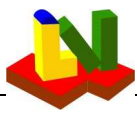
Remark

#### Openings according to UG-39

Nozzle material		
Opening diameter, corroded ( $\leq d/2$ )	$d_i$	in
Nozzle wall thickness without allowances	$t_n$	in
Allowable nozzle stress	$S_n$	psi
Wall thickness reserve	$t'$	<b>-0.2145</b> in
Available reinforcement area (plate)	$A_1$	in <sup>2</sup>
Required reinforcement area	$A$	in <sup>2</sup>
Alternative plate thickness acc. UG-39(d) corroded	$t_A$	in
Remark		

Allowable unreinforced opening diameter  $d$  for welded, brazed, and flued connections acc. UG 36(c)3

$d \leq 89 \text{ mm}$ for $t \leq 10 \text{ mm}$	or	$d \leq 3 \frac{1}{2} \text{ in}$ for $t \leq \frac{3}{8} \text{ in}$
$d \leq 60 \text{ mm}$ for $t > 10 \text{ mm}$	or	$d \leq 2 \frac{3}{8} \text{ in}$ for $t > \frac{3}{8} \text{ in}$



**Equations**

$$t_E = d \cdot \sqrt{1.9 \cdot W_E \cdot \frac{h_g}{(S_E \cdot E \cdot d^3)}} = 749.3 \text{ mm} \cdot \sqrt{1.9 \cdot 1054673 \text{ N} \cdot \frac{22.23 \text{ mm}}{(138 \text{ N/mm}^2 \cdot 1 \cdot (749.3 \text{ mm})^3)}} = 20.75 \text{ mm}$$

$$t_B = d \cdot \sqrt{C \cdot \frac{P_0}{(S_B \cdot E)} + 1.9 \cdot W_{m1} \cdot \frac{h_g}{(S_B \cdot E \cdot d^3)}} =$$

$$749.3 \text{ mm} \cdot \sqrt{0.3 \cdot \frac{9.308 \text{ bar}}{(122.8 \text{ N/mm}^2 \cdot 1)} + 1.9 \cdot 493841 \text{ N} \cdot \frac{22.23 \text{ mm}}{(122.8 \text{ N/mm}^2 \cdot 1 \cdot (749.3 \text{ mm})^3)}} = 38.77 \text{ mm}$$

$$38.77 \text{ mm} = \text{Max} \begin{cases} t_E \\ t_B \end{cases}$$

$$t_m = d \cdot \sqrt{1.9 \cdot \max \left( \frac{W_E}{S_E}, \frac{W_{m1}}{S_B} \right) \cdot \frac{h_g}{(E \cdot d^3)}} = 749.3 \text{ mm} \cdot \sqrt{1.9 \cdot 7643 \text{ mm}^2 \cdot \frac{22.23 \text{ mm}}{(1 \cdot (749.3 \text{ mm})^3)}} = 20.75 \text{ mm}$$

$$t' = E_1 \cdot (t_h - c_1 - c_2) - t_{(E=1)} = 1 \cdot (36.5 \text{ mm} - 0 \text{ mm} - 3.175 \text{ mm}) - 38.77 \text{ mm} = -5.448 \text{ mm}$$

Available reinforcement area analogously to Fig. UG-37.1

If

$$d_i > 2 \cdot (t_0 + t_n) \Leftrightarrow d_i > 2 \cdot (33.32 \text{ mm} + t_n)$$

Fig. UG-37.1

then

$$A_1 = \left( d_i - 2 \cdot t_n \cdot \left( 1 - \frac{S_n}{S_B} \right) \right) \cdot t' = \left( d_i - 2 \cdot t_n \cdot \left( 1 - \frac{S_n}{122.8 \text{ N/mm}^2} \right) \right) \cdot -5.448 \text{ mm} = A_1$$

Fig. UG-37.1

else

$$A_1 = 2 \cdot \left[ t_0 + t_n - t_n \cdot \left( 1 - \frac{S_n}{S_B} \right) \right] \cdot t' =$$

$$2 \cdot \left[ 33.32 \text{ mm} + t_n - t_n \cdot \left( 1 - \frac{S_n}{122.8 \text{ N/mm}^2} \right) \right] \cdot -5.448 \text{ mm} = A_1$$

Fig. UG-37.1

Required reinforcement area acc. UG-39(b)(1)

$$A = 0.5 \cdot t \cdot d_i + t \cdot t_n \cdot \left( 1 - \frac{S_n}{S_B} \right) = 0.5 \cdot 38.77 \text{ mm} \cdot d_i + 38.77 \text{ mm} \cdot t_n \cdot \left( 1 - \frac{S_n}{122.8 \text{ N/mm}^2} \right) = A$$

UG-39 (b)  
(1)

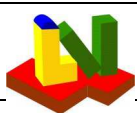
If  $A_1 > A$  in<sup>2</sup> > in<sup>2</sup> is not met, the available reinforcement area can better be calculated acc. UG-37 analogously to openings in cylinders (Longitudinal plane, F=1)

$$A_{avl} \quad \text{in}^2 \quad \text{acc. UG-37} \quad (\geq A \quad \text{in}^2)$$

Alternatively the plate thickness without allowances can be increased

$$t \quad \text{in} \quad \text{acc. UG-39(d)} \quad (\leq t_0 \quad 1.312 \text{ in})$$





## E.4.6.2 - Unstayed flat heads and covers - ASME BPVC VIII-1 UG-34 & UG-39: 2017

### Non-circular flat heads and plates without flange moment

#### Design data

Design pressure	$p_D$	400	psi
Hydrostatic head	$D_p$	0	psi
Calculation pressure	$p_0$	400	psi
Calculation temperature	$T_0$	500	°F
Design type (Fig. UG-34)	Type	c	

#### Cylinder

Outside diameter	$D_0$	in
Final thickness without allowance	$t_s$	in
Required thickness without allowance	$t_r$	in
Final thickness for type b1 ( $\geq 2 \cdot t_s$ )	$t_f$	in

#### Flat head or plate

Final wall thickness	$t_h$	0.8	in
Wall thickness allowance	$c_1$	0	in
Allowance (corrosion)	$c_2$	0.125	in
Wall thickness without allowances	$t_0$	0.675	in
Short span	$d$	7.375	in
Long span	$D$	9.5	in
Joint efficiency	$E$	1	

#### Material data

Material	K02700-SA-516-70-Class:-Size:		
Allowable stress	$S$	19957	psi

#### Results

Ratio	$m$		
Design factor	$Z$	1.537	
Design factor	$C$	0.2	
Required thickness	$t$	0.5789	in
Allowable excess pressure	$P$	543.9	psi
Required thickness incl. allowances	$t + c_1 + c_{2 < 7 \text{sub} >}$	0.7039	in
Required bend radius	$r_{\min}$		in

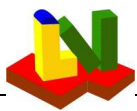
Remark

#### Openings according to UG-39

Nozzle material			
Opening diameter, corroded	$d_i$	in	$\leq d/2$
Nozzle wall thickness without allowances	$t_n$	in	
Allowable nozzle stress	$S_n$	psi	
Wall thickness reserve	$t'$	0.09614	in
Available reinforcement area (plate)	$A_1$	in <sup>2</sup>	
Required reinforcement area	$A$	in <sup>2</sup>	
Alternative plate thickness acc. UG-39(d) corroded	$t_A$	in	
Remark			

Allowable unreinforced opening diameter  $d$  for welded, brazed, and flued connections acc. UG 36(c)3

$d \leq 89 \text{ mm for } t \leq 10 \text{ mm}$	or	$d \leq 3 \frac{1}{2} \text{ in for } t \leq \frac{3}{8} \text{ in}$
$d \leq 60 \text{ mm for } t > 10 \text{ mm}$	or	$d \leq 2 \frac{3}{8} \text{ in for } t > \frac{3}{8} \text{ in}$



### Equations

$$m = \frac{t_r}{t_s}$$

$$Z = 3.4 - 2.4 \cdot \frac{d}{D} = 3.4 - 2.4 \cdot \frac{187.3 \text{ mm}}{241.3 \text{ mm}} = 1.537$$

$$1.537 \leq 2.5$$

$$t = d \cdot \sqrt{Z \cdot C \cdot \frac{P_0}{(S \cdot E)}} = 187.3 \text{ mm} \cdot \sqrt{1.537 \cdot 0.2 \cdot \frac{27.58 \text{ bar}}{(137.6 \text{ N/mm}^2 \cdot 1)}} = 14.7 \text{ mm}$$

UG-34 (b-2)  
(3)

$$t' = E_1 \cdot (t_h - c_1 - c_2) - t_{(E=1)} = 1 \cdot (20.32 \text{ mm} - 0 \text{ mm} - 3.175 \text{ mm}) - 14.7 \text{ mm} = 2.442 \text{ mm}$$

Available reinforcement area analogously to Fig. UG-37.1

If

$$d_i > 2 \cdot (t_0 + t_n) \Leftrightarrow d_i > 2 \cdot (17.15 \text{ mm} + t_n)$$

Fig. UG-37.1

then

$$A_1 = \left[ D_i - 2 \cdot t_n \cdot \left( 1 - \frac{S_n}{S_B} \right) \right] \cdot t' = \left[ D_i - 2 \cdot t_n \cdot \left( 1 - \frac{S_n}{137.6 \text{ N/mm}^2} \right) \right] \cdot 2.442 \text{ mm} = A_1$$

Fig. UG-37.1

else

$$A_1 = 2 \cdot \left[ t_0 + t_n - t_n \cdot \left( 1 - \frac{S_n}{S_B} \right) \right] \cdot t' =$$

$$2 \cdot \left[ 17.15 \text{ mm} + t_n - t_n \cdot \left( 1 - \frac{S_n}{137.6 \text{ N/mm}^2} \right) \right] \cdot 2.442 \text{ mm} = A_1$$

Fig. UG-37.1

Required reinforcement area acc. UG-39(b)(1)

$$A = 0.5 \cdot t \cdot d_i + t \cdot t_n \cdot \left( 1 - \frac{S_n}{S_B} \right) = 0.5 \cdot 14.7 \text{ mm} \cdot d_i + 14.7 \text{ mm} \cdot t_n \cdot \left( 1 - \frac{S_n}{137.6 \text{ N/mm}^2} \right) = A$$

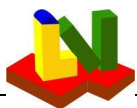
UG-39 (b)  
(1)

If  $A_1 > A$  in<sup>2</sup> > in<sup>2</sup> is not met, the available reinforcement area can better be calculated acc. UG-37 analogously to openings in cylinders (Longitudinal plane, F=1)

$A_{avl}$  in<sup>2</sup> acc. UG-37 (  $\geq A$  ) in<sup>2</sup> )

Alternatively the plate thickness without allowances can be increased

$t$  in acc. UG-39(d) (  $\leq t_0$  ) **0.675** in )



# ASME BPVC VIII-1 2017

## Example E4.6.1 - E4.6.2 PTB-4-2013

### Appendix: Material documentation

Section 1: Flansch/E 4.16.1  
Section 2: Boden/Platte/E 4.6.1

#### Material specification:

Regulation: ASMET1A:2017Spec. No.: SA-105 Product: Forgings  
Material code: K03504-SA-105--Class:-Size: Short name: Carbon steel

#### Design conditions and dimensions:

Temperature [°C]: 343,3333 Pressure [bar]: 9,30798  
Thickness [mm]: 3 Outside diameter [mm]: 0

#### Material values for test and design conditions:

	Test condition	Operating condition
Nominal design strength [N/mm²]:	138.00	122.80
Safety factor:		
Allowable stress [N/mm²]:	138.00	122.80
Modulus of elasticity [kN/mm²]:	201,3	178,6667

Wall thickness tolerance [mm]: 0.00 acc. to SA-105

#### Notes:

G10 General Requirements

Upon prolonged exposure to temperatures above 425°C, the carbide phase of carbon steel may be converted to graphite. See Nonmandatory Appendix A, A-201 and A-202.

#### S1 Size Requirements

For Section I applications, stress values at temperatures of 450°C and above are permissible but, except for tubular products 75 mm O.D. or less enclosed within the boiler setting, use of these materials at these temperatures is not current practice.

#### T2 Time-Dependent Properties

Allowable stresses for temperatures of 400°C and above are values obtained from time-dependent properties.

--

Creep rupture strength for 100000 h [MPa]:

#### Tensile strength and yield stress at ambient temperature:

Diam./...	Tensile str...	ReH.....	Rupture.....	Rupture.....
Thick.....	Rm min.....	Rm max.....	elong.....	elong.....
<= mm.....	MPa.....	MPa.....	MPa.....	MPa.....
			lengt. %.....	lat. %.....

K-values as function of the temperature

Diam./...	50°C.....	100°C.....	150°C.....	200°C.....	250°C.....	300°C.....	350°C.....	400°C.....
Thick.....	MPa.....	MPa.....	MPa.....	MPa.....	MPa.....	MPa.....	MPa.....	MPa.....
<= mm.....								
	138.....	138.....	138.....	136.....	129.....	122.....	101.....	

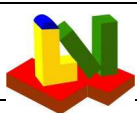
K-values as function of the temperature

Diam./...	450°C.....	500°C.....	550°C.....	600°C.....	650°C.....	700°C.....	800°C.....
Thick.....	MPa.....	MPa.....	MPa.....	MPa.....	MPa.....	MPa.....	MPa.....
<= mm.....							
	67.....	33.6.....	12.9.....				

#### Modulus of elasticity in dependence of the temperature:

Static modulus of elasticity in [kN/mm²] at the temperature of

-75..	-200..	-125..	25..	100..	150..	200..	250..	300..	350..	400..	450..	500..	550..	600..
207..	215..	211..	201..	197..	194..	191..	188..	183..	178..	170..	161..	149..	136..	121..



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## Example E4.6.1 - E4.6.2 PTB-4-2013

### Coefficient of linear expansion:

Thermal coefficient of expansion between 20°C and

Density (20 °C) kg/dm³	100°C	200°C	300°C	400°C	500°C	600°C	700°C	800°C	Heat cond.	Heat capac.
	10E-6/K	10E-6/K	10E-6/K	10E-6/K	10E-6/K	10E-6/K	10E-6/K	10E-6/K	W/Km	J/kgK
7.85	12,1	12,7	13,3	13,8	14,4	14,8	15,1	15,4		

Section 1: Schraube/E 4.16.1

### Material specification:

Regulation: ASMET3:2010Spec. No.: SA-193 Product: Bolting  
Material code: G41400-SA-193-B7-Class:-Size:<=64 Short name: 1Cr-0.2Mo

### Design conditions and dimensions:

Temperature [°C]: 343,3333 Pressure [bar]: 9,30798  
Thickness [mm]: 3 Outside diameter [mm]: 0

### Material values for test and design conditions:

	Test condition	Operating condition
Nominal design strength [N/mm²]:	172.00	172.00
Safety factor:		
Allowable stress [N/mm²]:	172.00	172.00
Modulus of elasticity [kN/mm²]:	204,3	183,4
Wall thickness tolerance [mm]:	0.00	acc. to SA-193
Notes:		

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Creep rupture strength for 100000 h [MPa]:

### Tensile strength and yield stress at ambient temperature:

Diam./	Tensile str.	ReH	Rupture	Rupture
Thick.	Rm min	Rm max	elong	elong
<= mm	MPa	MPa	lngt. %	lat. %

K-values as function of the temperature

Diam./	50°C	100°C	150°C	200°C	250°C	300°C	350°C	400°C
Thickn.	MPa	MPa	MPa	MPa	MPa	MPa	MPa	MPa
<= mm								
	172	172	172	172	172	172	172	162

K-values as function of the temperature

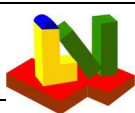
Diam./	450°C	500°C	550°C	600°C	650°C	700°C	800°C
Thickn.	MPa	MPa	MPa	MPa	MPa	MPa	MPa
<= mm							
	118	68.8	18.9				

### Modulus of elasticity in dependence of the temperature:

Static modulus of elasticity in [kN/mm²] at the temperature of

650	-75	-200	-125	25	100	150	200	250	300	350	400	450	500	550
150	210	218	213	204	200	197	193	190	186	183	179	174	169	164

Static modulus of elasticity in [kN/mm²] at the temperature of



# ASME BPVC VIII-1 2017

## Example E4.6.1 - E4.6.2 PTB-4-2013

600.....|700.....|.....|.....|.....  
-----+-----+-----+-----+-----  
157.....|142.....|.....|.....|.....

### Coefficient of linear expansion:

Thermal coefficient of expansion between 20°C and

Density|100°C..|200°C..|300°C..|400°C..|500°C..|600°C..|700°C..|800°C..|Heat...|Heat...  
(20 °C)|.....|.....|.....|.....|.....|.....|.....|.....|cond...|capac..  
kg/dm³·|10E-6/K|10E-6/K|10E-6/K|10E-6/K|10E-6/K|10E-6/K|10E-6/K|10E-6/K|W/Km...|J/kgK..  
-----+-----+-----+-----+-----+-----+-----+-----+-----+-----  
7.85...|12,1...|12,7...|13,3...|13,8...|14,4...|14.8...|15.1...|15.4...|.....|.....  
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Section 3: Boden/Platte/E.4.6.2

### Material specification:

Regulation: ASMET1A:2017Spec. No.: SA-516 Product: Plate  
Material code: K02700-SA-516-70-Class:-Size: Short name: Carbon steel

### Design conditions and dimensions:

Temperature [°C]: 260 Pressure [bar]: 27,5792  
Thickness [mm]: 3 Outside diameter [mm]: 0

### Material values for test and design conditions:

	Test condition	Operating condition
Nominal design strength [N/mm²]:	138.00	137.60
Safety factor:		
Allowable stress [N/mm²]:	138.00	137.60
Modulus of elasticity [kN/mm²]:	202,35	188,2

Wall thickness tolerance [mm]: 0.00 acc. to SA-516

Notes:

G10 General Requirements

Upon prolonged exposure to temperatures above 425°C, the carbide phase of carbon steel may be converted to graphite. See Nonmandatory Appendix A, A-201 and A-202.

S1 Size Requirements

For Section I applications, stress values at temperatures of 450°C and above are permissible but, except for tubular products 75 mm O.D. or less enclosed within the boiler setting, use of these materials at these temperatures is not current practice.

T2 Time-Dependent Properties

Allowable stresses for temperatures of 400°C and above are values obtained from time-dependent properties.

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Creep rupture strength for 100000 h [MPa]:

### Tensile strength and yield stress at ambient temperature:

Diam./	Tensile str.	ReH	Rupture	Rupture
Thick.....	Rm min.....	Rm max.....	elong.....	elong.....
<= mm.....	MPa.....	MPa.....	lngt. %.....	lat. %.....
-----+-----+-----+-----+-----				
..... ..... ..... ..... .....				

K-values as function of the temperature

Diam./	50°C	100°C	150°C	200°C	250°C	300°C	350°C	400°C
Thickn..	MPa	MPa	MPa	MPa	MPa	MPa	MPa	MPa
<= mm	-----+-----+-----+-----+-----+-----+-----+-----							
.....	..... 138..... 138..... 138..... 138..... 136..... 128..... 101.....							

K-values as function of the temperature

Diam./	450°C	500°C	550°C	600°C	650°C	700°C	800°C
Thickn..	.....	.....	.....	.....	.....	.....	.....

